

Introduction to LOCA

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LOCA Modelling

- One of the main application of choked flows is in depressurisation of pressurised vessels due to breaks in pipes associated with the system
- In reactors these kind of accidents are called Loss of Coolant Accidents (LOCA)
- Analysis of such accidents are very complex. But for illustrative purposes, simple lumped depressurisation is discussed in this lecture
- We shall first discuss single-phase cases and then move on to two-phase cases

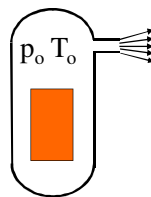
Single-Phase Depressurisation-I

- The total break area is A
- For single-phase case, the choked mass flow rate was obtained in the lecture on choking as

$$\Rightarrow \dot{m} = A \rho_o \left[2 c_p T_o \left(\left(\frac{p}{p_o} \right)^{\frac{2}{\gamma}} - \left(\frac{p}{p_o} \right)^{\frac{\gamma+1}{\gamma}} \right) \right]^{0.5} \quad (1)$$

- Under choked conditions, it was shown that the critical pressure ratio was given by

$$\Rightarrow \frac{p}{p_o} = \left(\frac{2}{\gamma+1} \right)^{\frac{\gamma-1}{\gamma}} \quad (2)$$



Single-Phase Depressurisation-II

- Plugging Eq. (2) in Eq. (1) and simplification gives

$$\begin{aligned} \Rightarrow \dot{m} &= A \left[\rho_o^2 2 c_p T_o \left(\frac{2}{\gamma+1} \right)^{\frac{\gamma+1}{\gamma-1}} \left(\frac{\gamma-1}{2} \right) \right]^{0.5} \\ &= A \left[\rho_o^2 2 c_p \frac{p_o}{\rho_o R} \left(\frac{2}{\gamma+1} \right)^{\frac{\gamma+1}{\gamma-1}} \left(\frac{\gamma-1}{2} \right) \right]^{0.5} \end{aligned}$$

- Expressing c_p/R in terms of γ , the above equation can be simplified as

$$= A [p_o \rho_o C]^{0.5} \quad \text{where, } C = \gamma \left(\frac{2}{\gamma+1} \right)^{\frac{\gamma+1}{\gamma-1}} \quad (3)$$

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Single-Phase Depressurisation-III

- Writing the mass balance for the fluid in the vessel

Mass Balance

$$\frac{d(V\rho)}{dt} = -\dot{m}$$

$$\Rightarrow V \frac{d(\rho)}{dt} = -\dot{m} \Rightarrow V \frac{d}{dt} \left(\frac{p}{RT} \right) = -\dot{m}$$

- Treating the instantaneous pressure and density in the vessel as the stagnation pressure, and using Eq. (3) for mass flow rate, we can write

$$\Rightarrow \frac{V}{R} \frac{d}{dt} \left(\frac{p}{T} \right) = -A \sqrt{p\rho C}$$

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Single-Phase Depressurisation-IV

- Writing the energy balance for the fluid in the vessel

Energy Balance

$$\frac{d(V\rho c_v T)}{dt} = \dot{Q} - \dot{m}h$$

$$\frac{d}{dt} V \frac{p}{RT} c_v T = \dot{Q} - \dot{m} c_p T$$

$$\frac{V c_v}{R} \frac{dp}{dt} = \dot{Q} - \dot{m} c_p T$$

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- Eqs. (4) and (5) are two ODE's which can be solved numerically with the initial conditions,

$$p = p_o, \quad T = T_o \quad \text{at } t = 0$$

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Single-Phase Depressurisation-V

- We can obtain analytical solutions under some special cases
- If the heat generated is such that $T = \text{constant}$, then we can say $T = T_o$. In such a case energy equation will not be required

Mass Balance

$$\frac{V}{RT_o} \frac{dp}{dt} = -A \sqrt{p \frac{p}{RT_o}} C$$

$$\Rightarrow \frac{dp}{dt} = -\frac{p}{\tau} \quad \text{where, } \tau = \frac{V}{A \sqrt{C R T_o}} \quad \Rightarrow p = p_o e^{-t/\tau}$$

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Single-Phase Depressurisation-VI

- Similarly, we can also obtain an analytical solution for the adiabatic case by employing isentropic law

$$\Rightarrow T = T_o \left(\frac{p_o}{p} \right)^{1-\gamma/\gamma}$$

- Substituting in the mass balance equation

$$\frac{V}{R} \frac{d}{dt} \left(\frac{p}{T} \right) = -A \sqrt{p\rho C}$$

$$\frac{V}{R} \frac{d}{dt} \left\langle \frac{p}{T_o \left\langle \frac{p_o}{p} \right\rangle^{1-\gamma/\gamma}} \right\rangle = -A \sqrt{\frac{p^2}{RT}} C = -A p \sqrt{\frac{C}{RT_o \left\langle \frac{p_o}{p} \right\rangle^{1-\gamma/\gamma}}}$$

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Single-Phase Depressurisation-VII

$$\Rightarrow \frac{V}{RT_o(p_o)^{1-\gamma/\gamma}} \frac{d}{dt} \left(p^{1+1-\gamma/\gamma} \right) = -Ap^{1+1-\gamma/2\gamma} \sqrt{\frac{C}{RT_o(p_o)^{1-\gamma/\gamma}}}$$

$$\Rightarrow \frac{d}{dt} \left(p^{1/\gamma} \right) = -\frac{A}{V} p^{1/2\gamma} \sqrt{CRT_o(p_o)^{1-\gamma/\gamma}}$$

$$\Rightarrow \frac{1}{\gamma} p^{1/\gamma-1} \frac{dp}{dt} = -\frac{A}{V} p^{1/2\gamma} \sqrt{CRT_o(p_o)^{1-\gamma/\gamma}}$$

$$\Rightarrow p^{\frac{2-2\gamma-(\gamma+1)}{2\gamma}} \frac{dp}{dt} = -\frac{\gamma}{\tau} (p_o)^{1-\gamma/2\gamma} \Rightarrow p^{\frac{1-3\gamma}{2\gamma}} \frac{dp}{dt} = -\frac{\gamma}{\tau} (p_o)^{1-\gamma/2\gamma}$$

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Single-Phase Depressurisation-VIII

- Integration of the above equation gives

$$\Rightarrow \frac{2\gamma}{1-\gamma} p^{\frac{1-\gamma}{2\gamma}} \Big|_{p_0}^p = -\frac{\gamma}{\tau} (p_o)^{1-\gamma/2\gamma} t$$

$$\Rightarrow \left(p^{\frac{1-\gamma}{2\gamma}} - p_0^{\frac{1-\gamma}{2\gamma}} \right) = (p_o)^{1-\gamma/2\gamma} \frac{\gamma-1}{2} \frac{t}{\tau}$$

$$\Rightarrow p^{\frac{1-\gamma}{2\gamma}} = (p_o)^{1-\gamma/2\gamma} \left(1 + \frac{\gamma-1}{2} \frac{t}{\tau} \right)$$

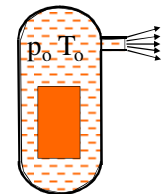
$$\Rightarrow p = p_o \left[\left(1 + \frac{\gamma-1}{2} \frac{t}{\tau} \right)^{\frac{2\gamma}{\gamma-1}} \right]$$

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Two-Phase Depressurisation-I

- In water reactors, as the depressurisation is initiated, two-phase conditions are encountered.
- The choking condition would also involve two-phase choking concepts discussed earlier
- Three models are considered for discussion:
 - Homogeneous Fluid Model
 - Separated Fluid Model
 - Bubble Rise Model

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Two-Phase Depressurisation-II

- First let us consider the homogeneous model, which is valid when the depressurisation is rapid
- The fluid is treated as homogeneous



Mass Balance $V \frac{d\bar{p}}{dt} = -\dot{m} \quad 1$

Energy Balance $V \frac{d}{dt} \left\{ \bar{p} \left(\bar{h} - \frac{p}{\bar{p}} \right) \right\} = -\dot{m} \bar{h} + \dot{Q} \quad 2$

$$V \left[-\frac{dp}{dt} + \bar{p} \frac{d\bar{h}}{dt} + \bar{h} \frac{d\bar{p}}{dt} \right] = -\dot{m} \bar{h} + \dot{Q}$$

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Two-Phase Depressurisation-III

- From Eqs. (1) and (2), we get

$$V \left[-\frac{dp}{dt} + \bar{\rho} \frac{d\bar{h}}{dt} - \bar{h} \frac{\dot{m}}{V} \right] = -\dot{m} \bar{h} + \dot{Q}$$

$$\Rightarrow \frac{d\bar{h}}{dt} - \frac{1}{\bar{\rho}} \frac{dp}{dt} = \frac{\dot{Q}}{V\bar{\rho}}$$

- Eq. (1) may be written as

$$V \frac{d(1/\bar{v})}{dt} = -\dot{m} \Rightarrow \frac{1}{\bar{v}^2} \frac{d\bar{v}}{dt} = -\frac{\dot{m}}{V}$$

$$\Rightarrow \frac{d(v_f + x v_{fg})}{dt} = -\frac{\dot{m}}{V} \bar{v}^2$$

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Two-Phase Depressurisation-IV

- As v_f and v_g are functions of p we can use chain rule to write,

$$\frac{dp}{dt} \left(\frac{dv_f}{dp} + x \frac{dv_{fg}}{dp} \right) + v_{fg} \frac{dx}{dt} = -\frac{\dot{m}}{V} \bar{v}^2$$

- Similarly, Eq. (3) may be written as

$$\frac{dp}{dt} \left(\frac{dh_f}{dp} + x \frac{dh_{fg}}{dp} - \bar{v} \right) + h_{fg} \frac{dx}{dt} = \frac{\dot{Q}}{V} \bar{v}$$

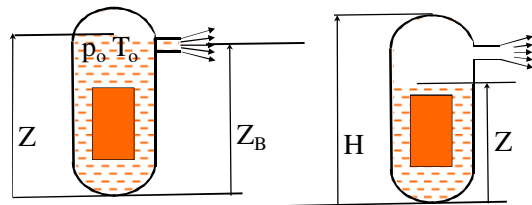
- Eqs. (4) and (5) can be modified to get explicit expressions for dp/dt and dx/dt and these can be solved numerically by ODE solvers

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Two-Phase Depressurisation-V

- Now let us consider the separated model, which is more of a conceptual model



Two-Phase Blowdown
 $Z > Z_B$

Single-Phase Blowdown
 $Z < Z_B$

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Two-Phase Depressurisation-VI

- The overall lumped mass and energy balances are identical
- The difference is in the treatment of stagnation enthalpy of the fluid coming out of the break.
- This is modelled as follows

$$h_0 = h_f \text{ for } Z \geq Z_B$$

$$h_0 = h_g \text{ for } Z < Z_B$$

- Thus, we need to track the interface height.

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Two-Phase Depressurisation-VII

- We can continue to use the previous Eqs. (4) and (5) as the governing equations for the transient variation of pressure, p and vapour mass fraction, x
- The volume fraction α can be expressed as a function of mass fraction as

$$\alpha = \frac{l}{l + \left(\frac{l-x}{x} \right) \frac{\rho_g}{\rho_f}}$$

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- Further for a cylindrical vessel we can write

$$l - \alpha = \frac{Z}{H}$$

- Thus the transient variation of Z can be obtained

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Two-Phase Depressurisation-VIII

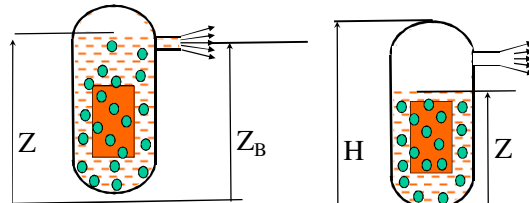
- The separated model described is not that realistic as vapour is generated in the liquid and has to be transported to the top space
- Hence there will always be some bubbles in the liquid.
- Now we will discuss a more realistic model called the bubble rise model
- It is fairly similar to the previous model but needs to distribute the steam in fluid appropriately
- In this model, we consider two regions
 - Steam Dome
 - Frothing Liquid with uniformly distributed bubbles

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Two-Phase Depressurisation-IX

- The bubbles are assumed to be of one diameter and are assumed to rise with a uniform velocity
- The blowdown modelling is similar and expressions for stagnation enthalpy are discussed later



Two-Phase Blowdown
 $Z > Z_B$

Single-Phase Blowdown
 $Z < Z_B$

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Two-Phase Depressurisation-X

- Let us define
 - Mass of liquid - M_l
 - Mass of vapour in bubbles - M_{gb}
 - Mass of vapour in steam dome - M_{sd}
 - Mass of steam in vessel - M_s
 - Average void fraction in the vessel - α
- The overall mass and energy balance continues to be the same as outlined in Eqs. (4) and (5). But tracking of interface height needs more elaborate book keeping.
- This is modelled as follows.

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Two-Phase Depressurisation-XI

$$M_s = A_v H \alpha \rho_g = M_{gb} + M_{sd} \quad (8)$$

$$M_{sd} = A_v (H - Z) \rho_g \quad (9)$$

$$M_{gb} = A_v H \alpha \rho_g - A_v (H - Z) \rho_g$$

$$\Rightarrow M_{gb} = A_v \rho_g (Z - (1 - \alpha)H) \quad (10)$$

- Further for the steam dome we can write

$$\frac{dM_{sd}}{dt} = \dot{m}_{gain} - \dot{m}_{loss} \quad (11)$$

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Two-Phase Depressurisation-XII

- The loss term in the previous equation can be modelled as

$$\dot{m}_{loss} = 0 \text{ for } Z \geq Z_B \quad (12)$$

$$\dot{m}_{loss} = \dot{m}_{break} \text{ for } Z < Z_B$$

- To model the steam added in the dome due to interface movement, consider a time Δt , during which interface moves by ΔZ

$$\text{mass gain in dome} = A_v \alpha_{gb} \left(V_b + \frac{dz}{dt} \right) \Delta t \rho_g$$

$$\Rightarrow \dot{m}_{gain} = A_v \alpha_{gb} \left(V_b + \frac{dz}{dt} \right) \rho_g \quad (13)$$

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Two-Phase Depressurisation-XIII

$$M_{gb} = \alpha_{gb} A_v Z \rho_g$$

$$\Rightarrow \dot{m}_{gain} = \frac{M_{gb}}{Z} \left(V_b + \frac{dz}{dt} \right) \quad (14)$$

- Substituting the expression for M_{gb} from Eq. (10) and neglecting dz/dt in comparison with V_b , we get

$$\dot{m}_{gain} = \frac{A_v \rho_g (Z - (1 - \alpha)H)}{Z} V_b \quad (15)$$

- Substituting the expression for $\dot{m}_{gain}$ from Eq. (17) in Eq. (11), we get

$$\frac{d(A_v (H - Z) \rho_g)}{dt} = A_v \rho_g \left(1 - (1 - \alpha) \frac{H}{Z} \right) V_b - \dot{m}_{loss}$$

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Two-Phase Depressurisation-XIV

$$\Rightarrow A_v (H - Z) \frac{d\rho_g}{dt} - A_v \rho_g \frac{(dZ)}{dt} = A_v \rho_g \left(1 - (1 - \alpha) \frac{H}{Z} \right) V_b - \dot{m}_{loss}$$

$$\Rightarrow \frac{dZ}{dt} = \frac{(H - Z)}{\rho_g} \frac{d\rho_g}{dp} \frac{dp}{dt} - \left(1 - (1 - \alpha) \frac{H}{Z} \right) V_b + \frac{\dot{m}_{loss}}{\rho_g A_v} \quad (16)$$

- Thus, the overall solution is obtained as follows
 - Eqs.(4) and (5) are solved to obtain p , x .
 - The overall α is obtained using is Eq. (7)
 - Suitable choking model is used. The definition of stagnation enthalpy is as follows

$$h_0 = \frac{M_l h_f + M_{gb} h_g}{M_l + M_{gb}} \text{ for } Z \geq Z_B$$

$$h_0 = h_g \text{ for } Z < Z_B$$

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- M_{gb} is obtained using Eq. (10)
- M_l can be obtained as

$$M_l = A_v H(1 - \alpha) \rho_f$$
- Finally, Z can be obtained by solving Eq. (16)
- Thus the entire solution can be obtained numerically