## **Assignment 6**

1. A small PWR plant operates at a power of 485 MW(th). The core is approximately 1.9 m in diameter and 2.3 m tall. It consists of 23,142 tubes of thickness 0.5 mm and inner diameter 7.5 mm on a square pitch of 10.5 mm. The tubes are filled with 3.4 w/o enriched UO<sub>2</sub>. The core is cooled with water entering at 260 °C flowing at a rate of 15.4 X  $10^6$  kg/hr at 150 bar. Compute the,

(a) average temperature of water leaving the core.

- (b) average power density in kW/litre, and
- (c) maximum linear heat rate, surface heat flux and volumetric heat generation rate.
- 2. Design a 3000 MW(th) bare cylindrical PWR, whose height to diameter ratio is 1. The fuel to coolant volume ratio is to be maintained equal to 1 in a square lattice. The system pressure and core inlet temperature shall be 175 bars and 285° C respectively. If the reactor has to work under the following three thermal constraints, viz., the
  - (i) maximum power density = 250 W/cc,
  - (ii) maximum clad surface heat flux =  $125 \text{ W/cm}^2$  and,
  - (iii) maximum core outlet temperature =  $325^{\circ}$  C,

Compute, the

(a) reactor dimensions and volume,

(b) fuel element diameter and lattice pitch,

(c) number of fuel elements, and

(d) mass flow rate and average coolant velocity.

You may assume:  $c_p$  of coolant = 6 kJ/kg-K and overall peaking factor = 3.64

Note: The N square lattices, when properly arranged forms a circular core of diameter D

3. The heat flux at the surface of fuel rod in an experimental reactor varies as shown in the figure. The fuel rods are 1 cm in diameter and 2 m long. Water (C =  $4.5 \text{ kJ/kg}^{\circ}$ C) enters at  $65^{\circ}$  C and flows at the rate of 160 kg/hr per rod. The heat transfer coefficient is 6000 W/m<sup>2</sup>-C. Calculate the temperature of coolant and fuel surface temperature at the middle and the exit of the channel.



4. A reactor is made up channels with fuel rods of 10 mm diameter. The associated coolant flow area is 1.5 cm<sup>2</sup>. The coolant enters at 10 Mpa at a velocity of 3.6 m/s and 250 °C. The exit temperature of the coolant is 295 °C. The axial power profile in the channel may be assumed to vary as,

 $\mathbf{q}^{''} = \mathbf{q}_{0}^{''} \mathbf{e}^{\frac{\pi z}{H}} \sin(\frac{\pi z}{H}),$ 

where H represents the height of the core, which is 3.6 m, z=0 represents the bottom of the core and  $q_0^{"}$  is a constant. Compute the location of the maximum fuel surface temperature, given the following:

 $\rho_{\rm f} = 732.3 \text{ kg/ m}^3$ ,

 $c_p$  of the coolant = 5.51 kJ/kg-K. Convective heat transfer coefficient = 10 kW/m<sup>2</sup>-K

$$\int e^{ax} \sin(bx) dx = \frac{e^{ax}}{a^2 + b^2} \left( a \sin(bx) - b \cos(bx) \right)$$

5 Consider a vertical fuel rod of 8 mm diameter wetted with a coolant mass flow of 0.5 kg/s. If the linear heat generation rate is as shown in the figure, and the coolant enters at 267 °C, compute the location at which the coolant will boil. You may assume that the saturated temperature of water at the fluid pressure is 285.8 °C,  $c_p = 5.444 \text{ kJ/kg-K}$ . If the heat transfer coefficient at that location is 8000 W/m<sup>2</sup>-K, compute the clad surface temperature.

